



Division of Strength of Materials and Structures

Faculty of Power and Aeronautical Engineering



Finite element method (FEM1)

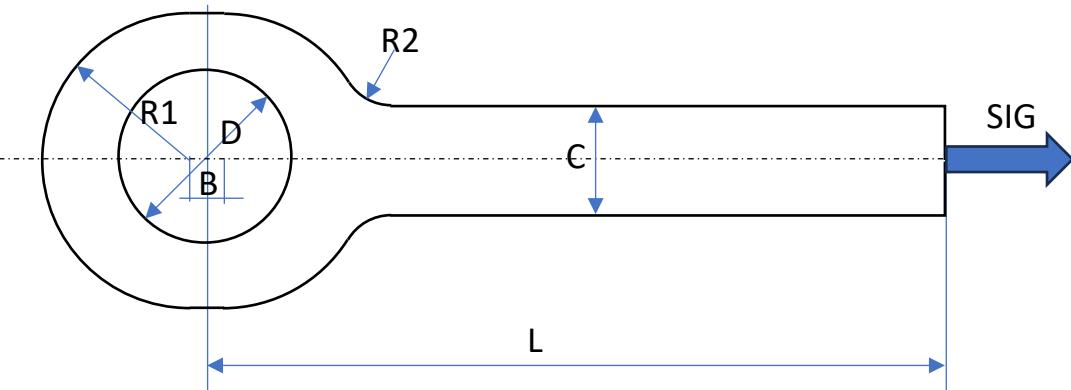
Lecture 2B. Example of a structural problem – pin connection

03.2025

Structural problem - modeling of the pin connection

Let us assume an infinitely stiff pin.

Data: $D=10\text{mm}$, $R1=8\text{mm}$, $R2=3\text{mm}$, $B=2\text{mm}$, $C=6\text{mm}$, $L=40\text{mm}$, $H=6\text{mm}$, $SIG=100\text{MPa}$, $E=2e5\text{MPa}$, $\nu=0.3$

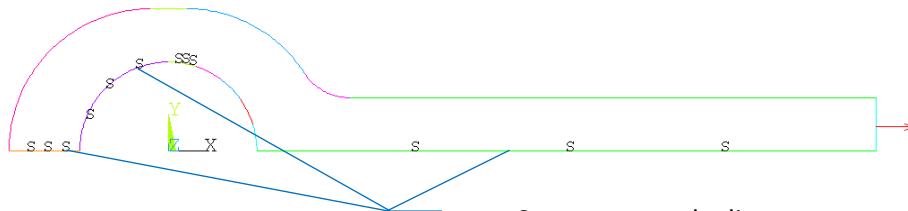
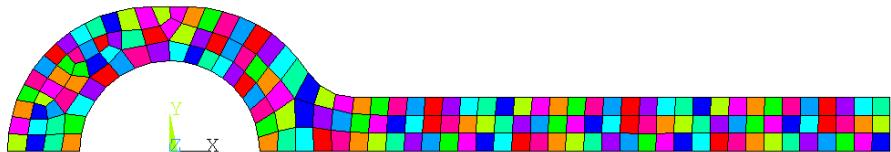


UCHO_2D.TXT

!Parameters:

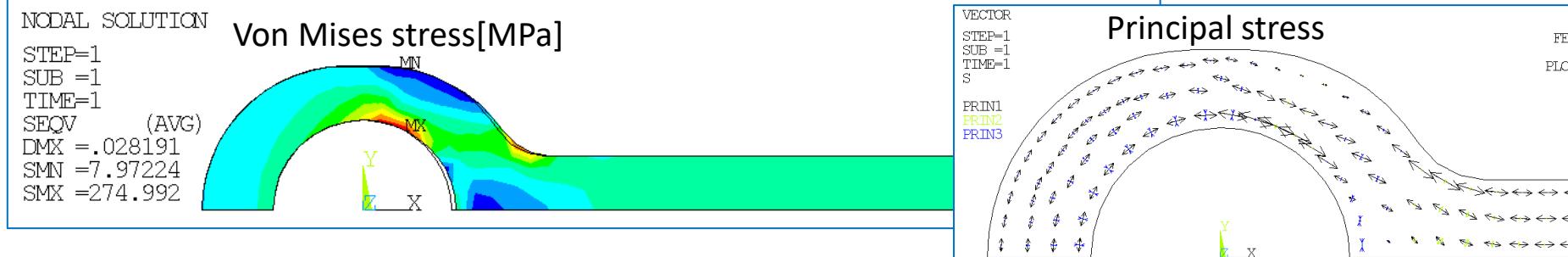
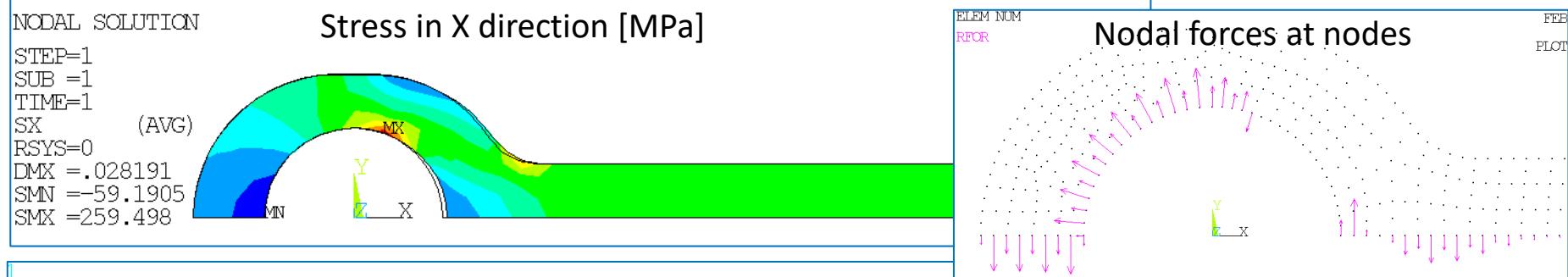
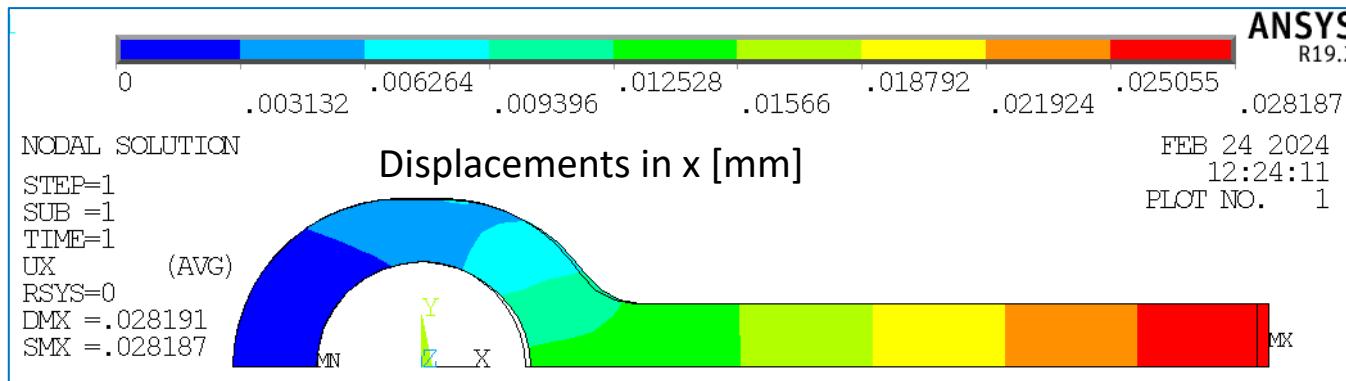
$D=10$! mm - pin diameter
 $R1=8$!mm- radius
 $R2=3$!mm- radius
 $B=2$!mm- lug elongation parameter
 $C=6$!mm- width of the rod
 $H=6$!mm- thickness of the rod
 $L=40$!mm- rod length from the center of the pin
 $SIG=100$!MPa- average stresses
 $E=2e5$!MPa- Young modulus
 $Nl=0.3$! Poisson ratio

ELDLUG=1 !mm- size of the element edge



Symmetry on the lines
(no displacements normal to the line)

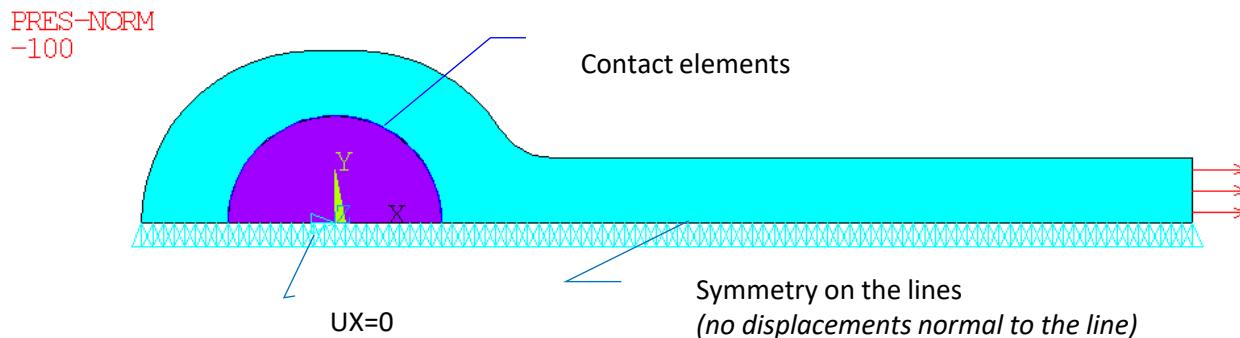
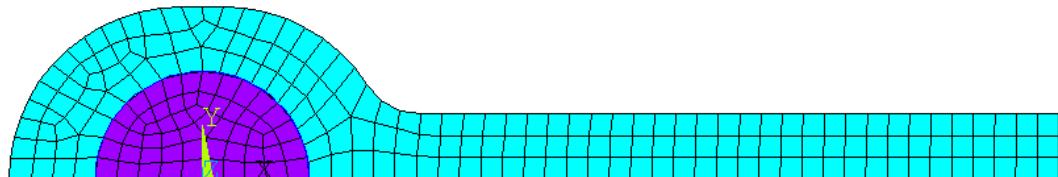
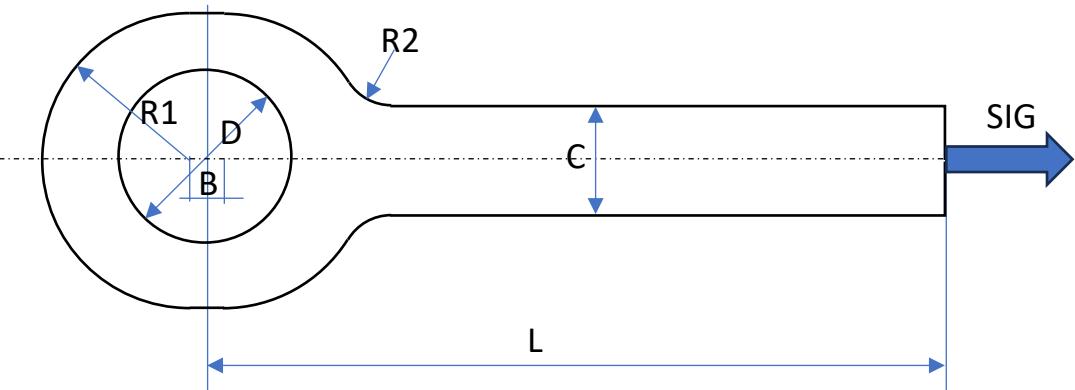
Shaping the lug of a bolt connection: simplified model **UCHO_2D**



Shaping the lug of a bolt connection: More accurate model ***UCHO_2D_contact_NL***

Steel Pin + Contact Elements.

Data: $D=10\text{mm}$, $R1=8\text{mm}$, $R2=3\text{mm}$, $B=2\text{mm}$, $C=6\text{mm}$, $L=40\text{mm}$, $H=6\text{mm}$, $SIG=100\text{MPa}$, $E=2e5\text{MPa}$, $\nu=0.3$



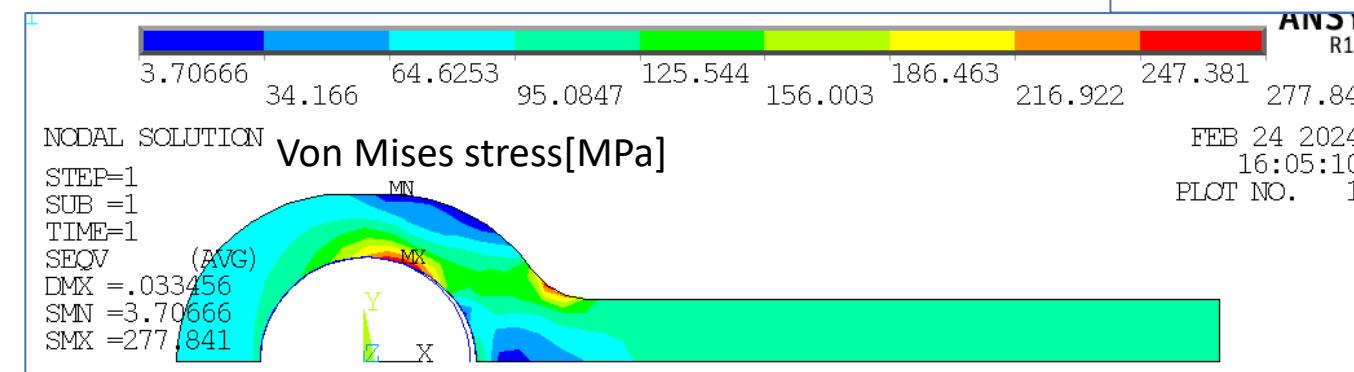
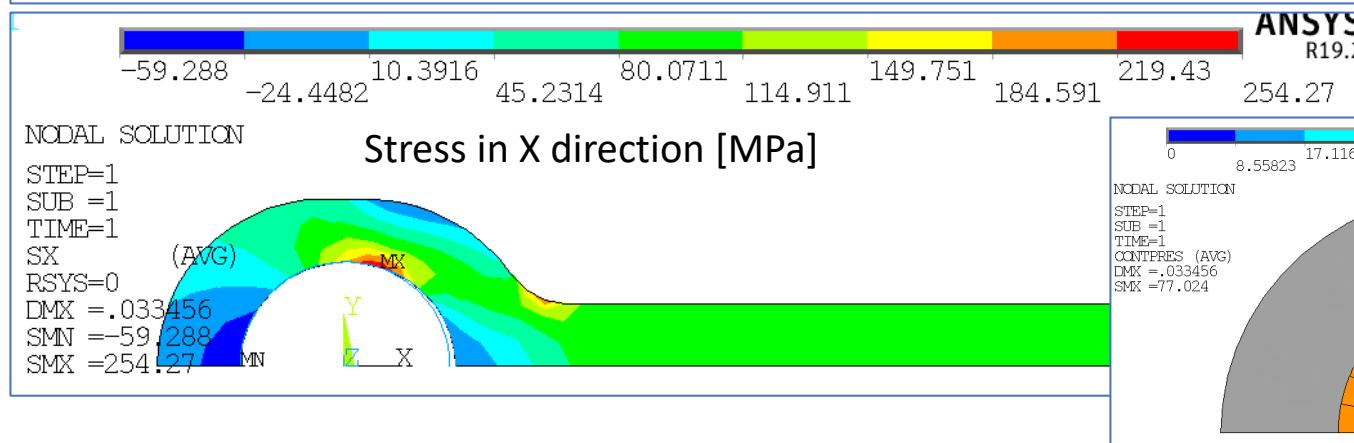
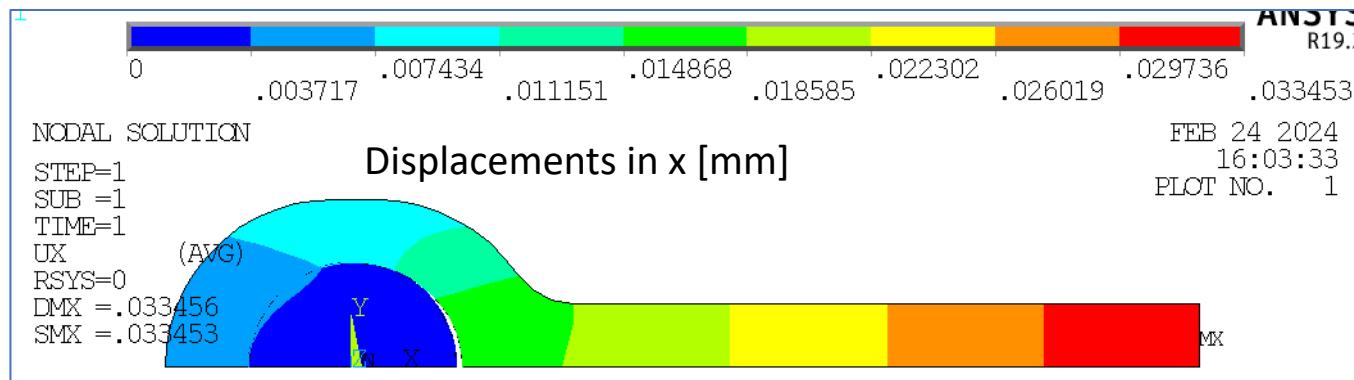
UCHO_2D_contact_NL.TXT

!Parameters:

$D=10$! mm - pin diameter
 $R1=8$!mm- radius
 $R2=3$!mm- radius
 $B=2$!mm- lug elongation parameter
 $C=6$!mm- width of the rod
 $H=6$!mm- thickness of the rod
 $L=40$!mm- rod length from the center of the pin
 $SIG=100$!MPa- average stresses
 $E=2e5$!MPa- Young modulus
 $Nl=0.3$! Poisson ratio

$ELDLUG=1$!mm- size of the element edge

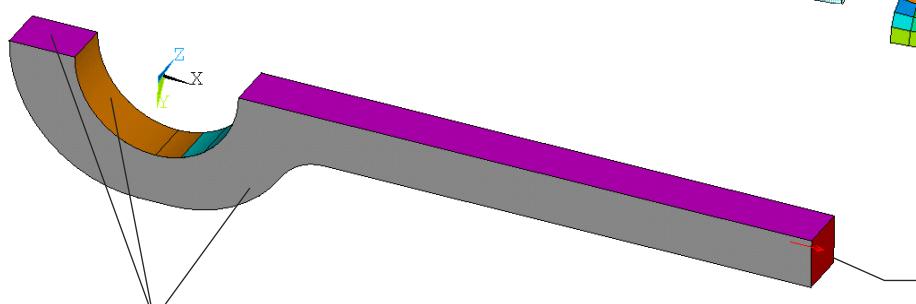
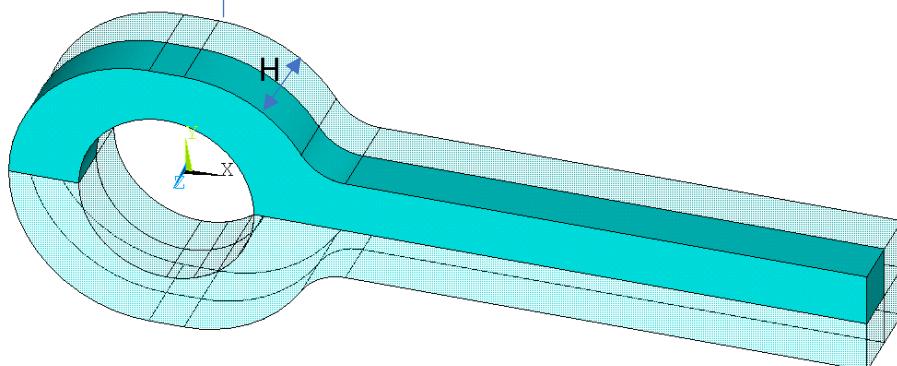
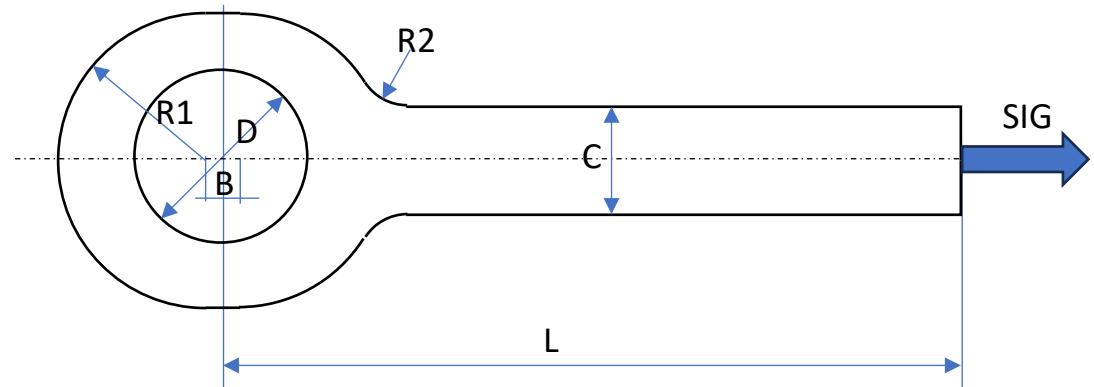
Shaping the lug of a bolt connection: More accurate model **UCHO_2D_contact NL**



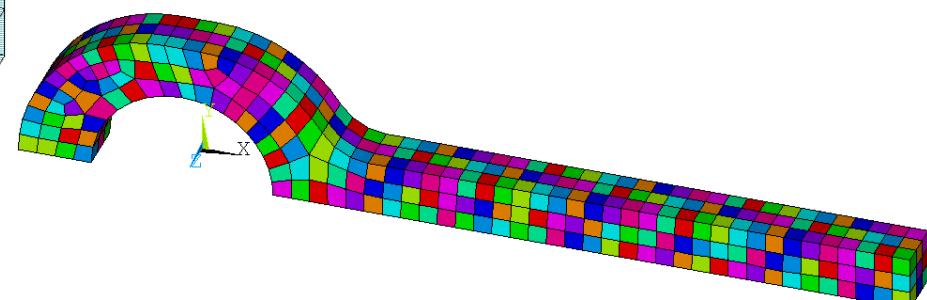
Shaping the lug of a bolt connection : simplified model **UCHO_3D**

Let us assume an infinitely stiff pin.

Dane: $D=10\text{mm}$, $R1=8\text{mm}$, $R2=3\text{mm}$, $B=2\text{mm}$, $C=6\text{mm}$, $L=40\text{mm}$, $H=6\text{mm}$, $SIG=100\text{MPa}$, $E=2e5\text{MPa}$, $\nu=0.3$



Symmetry on surfaces
(no displacements normal to the surface)



Negative pressure on the surface

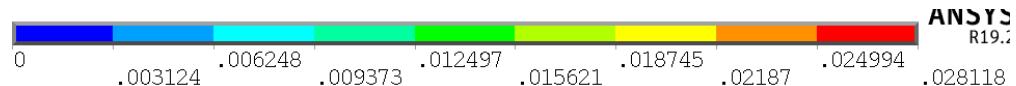
UCHO_3D.TXT

!Parameters:

$D=10$! mm - pin diameter
 $R1=8$!mm- radius
 $R2=3$!mm- radius
 $B=2$!mm- lug elongation parameter
 $C=6$!mm- width of the rod
 $H=6$!mm- thickness of the rod
 $L=40$!mm- rod length from the center of the pin
 $SIG=100$!MPa- average stresses
 $E=2e5$!MPa- Young modulus
 $\nu=0.3$! Poisson ratio

$ELDLUG=1$!mm- size of the element edge

Shaping the lug of a bolt connection : simplified model **UCHO_3D**



NODAL SOLUTION

STEP=1
SUB =1
TIME=1
UX
RSYS=0
DMX =.028125
SMX =.028118

Displacements in x [mm]



NODAL SOLUTION

STEP=1
SUB =1
TIME=1
SX
RSYS=0
DMX =.028125
SMN =-61.4849
SMX =272.371

Stress in X direction [MPa]



NODAL SOLUTION

STEP=1
SUB =1
TIME=1
SEQV
DMX =.028125
SMN =5.65869
SMX =278.975

Von Mises stress[MPa]

Principal stress

